

DESIGN AND CFD ANALYSIS OF HAIR PIN HEAT EXCHANGER AT DIFFERENT NANO-FLUIDS

¹ZAHID IMAM

IJMTARC - VOLUME - V - ISSUE - 19 - SEPT 2017

²P.N. KIRAN KUMAR

¹M.Tech, Thermal Engineering student, Department of Mechanical Engineering, Malla Reddy Engineering College (Autonomous), Dhulapally, Maisammaguda, Hyderabad-500100- India.

²Assistant Professor, Department of Mechanical Engineering, Malla Reddy Engineering College (Autonomous), Dhulapally, Maisammaguda, Hyderabad-500100- India.

Abstract:

In this thesis, different NANO fluids mixed with base fluid water are analyzed for their performance in the hair pin heat exchanger. The NANO fluids are Aluminum Oxide and Titanium carbide for two volume fractions 0.4, 0.5. Theoretical calculations are done determine the properties for NANO fluids and those properties are used as inputs for analysis.

Keywords: Finite element analysis, Hair pin heat exchanger, CFD analysis, thermal analysis.

I. INTRODUCTION

Heat exchangers are one of the mostly used equipment in the process industries. Heat Exchangers are used to transfer heat between two process streams. One can realize their usage that any process which involve cooling, heating, condensation, boiling or evaporation will require a heat exchanger for these purpose. Process fluids, usually are heated or cooled before the process or undergo a phase change. Different heat exchangers are named according to their application. For example, heat exchangers being used to condense is known as condensers, similarly heat exchanger for boiling purposes are called boilers. Performance and efficiency of heat exchangers are measured through the amount of heat transfer using least area of heat transfer and pressure drop. A better presentation of its efficiency is done by calculating over all heat transfer coefficient. Pressure drop and area required for a certain amount of heat transfer, provides an insight about the capital

cost and power requirements (Running cost) of a heat exchanger. Usually, there is lots of literature and theories to design a heat exchanger according to the requirements.



II. LITERATURE REVIEW

DESIGN AND ANAYSIS OF DOUBLE PIPE HEAT EXCHANGER USING COMPUTATIONAL METHOD Heat transfer equipment is defined by the function it fulfills in a process. On the similar path, Heat exchangers are the equipment used in industrial processes to recover heat between two process fluids. They are widely used in space heating, refrigeration, air conditioning, power plants, chemical plants, petrochemical plants, petroleum refineries, and natural gas processing. The operating efficiency of these exchangers plays a very key role in the overall running cost of a plant. So the designers are on a trend of developing heat



IJMTARC - VOLUME - V - ISSUE - 19 - SEPT 2017

ISSN: 2320-1363

exchangers which are highly efficient compact, and cost effective. A common problem in industries is to extract maximum heat from a utility stream coming out of a particular process, and to heat a process stream. Therefore the objective of present work involves study of refinery process and applies phenomena of heat transfer to a double pipe heat exchanger.

III. Dimensions of designed double tube Hair-pin heat exchanger:

Outer pipe specification Inner tube specification

Copper tube of U bends I.D. of shell= 19.05 mm I.D. of tube = 8.4 mm O.D. of shell = 22 mm O.D. of tube = 9.5 mm Center to center distance is taken Wall thickness= 0.55 mm

1.5 - 1.8 times of outer dia. of shell. Thermal conductivity of wall= 385 w/m2K Length of each G.I. pipe =22.86cm Effective length of copper tube through which heat transfer could take place= 45cm Total length of the copper tube = straight part (51cm) + U-shaped bend part (9cm) =60cm

3D model of hair pin heat exchanger







2D model of hair pin heat exchanger



IV. INTRODUCTION TO FINITE ELEMENT METHOD

FEM/FEA helps in evaluating complicated structures in a system during the planning stage. The strength and design of the model can be improved with the help of computers and FEA which justifies the cost of the analysis. FEA has prominently increased the design of the structures that were built many years ago.

INTRODUCTION TO CFD



IJMTARC - VOLUME - V - ISSUE - 19 - SEPT 2017

ISSN: 2320-1363

Computational fluid dynamics, usually abbreviated as CFD, is a branch of fluid mechanics that uses numerical methods and algorithms to solve and analyze problems that involve fluid flows.

CALCULATIONS TO DETERMINE PROPERTIES OF NANO FLUID BY CHANGING VOLUME FRACTIONS NANO FLUID CALCULATIONS NOMENCLATURE

 ρ_{nf} = Density of nano fluid (kg/m³)

 $\rho_s = \text{Density of solid material}$ (kg/m³)

 $\rho_{\rm w}$ = Density of fluid material (water) (kg/m³)

 ϕ = Volume fraction

 C_{pw} = Specific heat of fluid material (water) (j/kg-k)

 C_{ps} = Specific heat of solid material (j/kg-k)

 $\mu_w = \text{Viscosity of fluid (water)}$ (kg/m-s)

- K_w = Thermal conductivity of fluid material (water) (W/m-k)
- K_s = Thermal conductivity of solid material (W/m-k)

DENSITY OF NANO FLUID

 $\rho_{nf} = \phi \times \rho_s + \left[(1 \text{-} \phi) \times \rho_w \right]$

SPECIFIC HEAT OF NANO FLUID

 $C_{p nf} = \frac{\phi \times \rho s \times Cps + (1 - \phi)(\rho w \times Cpw)}{\phi \times \rho s + (1 - \phi) \times \rho w}$

VISCOSITY OF NANO FLUID

 $\mu_{nf} = \mu_w (1 + 2.5 \phi)$

THERMAL CONDUCTIVITY OF NANO FLUID

 $K_{nf} = \frac{K_{s+2Kw+2(Ks-Kw)(1+\beta)^{3}\times\phi}}{K_{s+2Kw-2(Ks-Kw)(1+\beta)^{3}\times\phi}} \times k_{w}$

NANO FLUID PROPERTIES

FLUD	Volume fraction	Thermal conductivity (w/m-k)	Specific heat (Mkg-k)	Density (kg/m²)	Viscosity (kg/m-s)	
ALUMINUM	8.4	2.647	1809	2150.92	0.002006	1
	0,5	4.17	1570.9	2439.1	0.002256	-
TITANIUM CARBIDE	0.4	2.625	\$357.01	2570.92	0.003006	1
	0.5	4.12	4059 1	2964.1	0.002256	1

CFD ANALYSIS OF HAIR PIN HEAT EXCHANGER

ALUMINUM OXIDE NANO FLUID

VOLUME FRACTION - 0.4

STATIC PRESSURE



VELOCITY MAGNITUDE







IJMTARC - VOLUME - V - ISSUE - 19 - SEPT 2017

ISSN: 2320-1363

MASS FLOW RATE



HEAT TRANSFER RATE



THERMAL ANALYSIS OF HAIR PIN HEAT EXCHANGER

MATERIAL-COPPER

IMPORTED MODEL



MESHED MODEL



BOUNDARY CONDITIONS

TEMPERATURE



HEAT FLUX



RATE THERMAL ANALYSIS OF HAIR PIN HEAT EXCHANGER

MATERIAL-ALUMINUM

TEMPERATURE



HEAT FLUX





IJMTARC – VOLUME – V – ISSUE - 19 – SEPT 2017

V. RESULT TABLES

CFD RESULT TABLES THERMAL ANALSIS RESULTS TABLES

Fluid	Pressure (Pa)	Velocity (m/s)	Heat transfer coefficient (wim2-k)	Mass flow rate(kg/s)	Heat transfer rate(W)
Water	6.46e-02	1.32e-02	1.50e+02	0.035707	7178.0925
Al2o3(&=0.4)	3.79e-02	6.31e-03	6.61e+02	0.05145	4613.6967
A12o3(d=0.5)	3.58e-02	5.64e-03	1.04e+03	0.0515709	4078.4819
TiC ()=0.4)	3.29e-02	5.30e-03	6.56e+02	0.0525819	14320.233
TiC((\$=0.5)	3.05e-02	4.62e-03	1.03e+03	0.058124	11510.686

Material	Temperature (⁰ C)		Heat flux(wim ²)	
	Min.	Max.		
Aluminum alloy	20.471	72.111	2.4941e7	
copper	19.28	72.316	2.7459e7	

CFD ANALYSIS GRAPHS

PRESSURE PLOT



VELOCITY PLOT



HEAT TRANSFER COEFFICIENT PLOT



MASS FLOW RATE PLOT





So it is able to be concluded the titanium carbide nano fluid at quantity fraction 0.5 fluid is the higher fluid for hair pin warmness exchanger and material is copper.

HEAT TRANSFER RATE PLOT



THERMAL ANALYSIS GRAPHS



VI. CONCLUSION

Hairpin Exchangers are available in unmarried tube (Double Pipe) or a couple of tubes within a hairpin shell (Multitude), naked tubes, finned tubes, U-tubes, straight tubes (with rod-through functionality), constant tube sheets and removable package deal.

By looking at the CFD analysis results the heat transfer coefficient at titanium carbide volume fraction 0.4.

By observing the thermal analysis consequences the heat flux cost more for copper material evaluated with aluminum alloy.

REFERENCES

1. A.O. Adelaja, S. J. Ojolo and M. G. Sobamowo, "Computer Aided Analysis of Thermal and Mechanical Design of Shell and Tube Heat Exchangers", Advanced Materials Vol. 367 (2012) pp 731-737 © (2012) Trans Tech Publications, Switzerland.

2. Yusuf Ali Kara, Ozbilen Guraras, "A computer program for designing of Shell and tube heat exchanger", Applied Thermal Engineering 24(2004) 1797–1805

3. Rajagapal THUNDIL KARUPPA RAJ and Srikanth GANNE, "Shell side numerical analysis of a shell and tube heat exchanger considering the effects of baffle inclination angle on fluid flow", Thundil Karuppa Raj, R., et al: Shell Side Numerical Analysis of a Shell and Tube Heat Exchanger ,THERMAL SCIENCE: Year 2012, Vol. 16, No. 4, pp. 1165-1174.

4. S. Noie Baghban, M. Moghiman and E. Salehi, " Thermal analysis of shell-side flow of shell-and tube heat exchanger using experimental and theoretical methods" (Received: October 1, 1998 -Accepted in Revised Form: June 3, 1999).

5. A.GopiChand, Prof.A.V.N.L.Sharma , G.Vijay Kumar, A.Srividya, "Thermal analysis of shell and tube heat exchanger using mat lab and floefd software", Volume: 1 Issue: 3 276 –281, ISSN: 2319 – 1163.

6. Hari Haran, Ravindra Reddy and Sreehari, "Thermal Analysis of Shell and Tube Heat Exchanger Using C and Ansys",International Journal of Computer Trends and Technology (IJCTT) – volume 4 Issue 7–July 2013.



IJMTARC - VOLUME - V - ISSUE - 19 - SEPT 2017

7. Donald Q.Kern. 1965. Process Heat transfer (23rdprinting 1986). McGraw-Hill companies.ISBN 0-07-Y85353-3.

8. Richard C. Byrne Secretary. 1968. Tubular Exchanger Manufacturers Association, INC. (8th Edition). 25 North Broadway Tarrytown, New York 10591.

9. R.H Perry. 1984. Perry's Chemical Engineer's Handbook (6th Edition Ed.). McGraw-Hill. ISBN 0-07-049479-7.

10. Ender Ozden, Ilker Tari, Shell Side CFD Analysis of A Small Shell And Tube Heat Exchanger, Middle East Technical University, 2010.